# **Leveraging Efficiency and Owner Value**

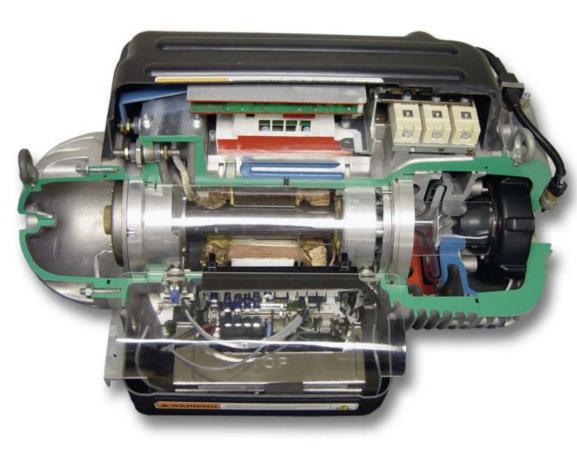
Latest Energy Studies - A Comparison by Compressor Types. Simple Energy, Effect of Oil, and Total Cost of Ownership Using Oil-Free Danfoss Turbocor Compressors.



# **Topics of Discussion**

- The Danfoss Turbocor Compressor Basic Owner Advantages
- Compressor and Chiller Controls Chiller or Plant?
- Correlating Geographic, Building Load and Compressor Profiles
- Water versus Air Cooled Chillers New paradigm emerging
- ASHRAE 90.1 2004 and 2010 Path A and B
- Loading Chillers Examples at 50% and elevated ChW Temps
- Building Type Energy Analysis Study and Conclusions
- High Cost of Oil Maintenance and Unsustainable U-Value
- Economizers and Adiabatic Assist for Air Cooled
- Reducing Noise and Raising Efficiency using Fan Diffusers
- Series-Counter-Flow Strategy and Effect on Lift
- Total Cost of Ownership Drivers

## What is the Turbocor Compressor?



• 100% Oil Free Centrifugal eliminates friction and oil system maintenance costs.

• R134a Refrigerant meets Montreal protocol and is not subject to phase-out.

 Magnetic Bearings eliminate frictional losses and greatly reduces maintenance

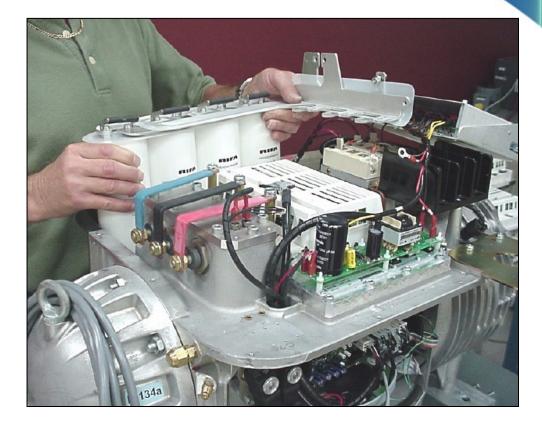
• SUSTAINABLE Efficiency as low as .30 kw/ton NPLV.

• Low 2-amp In-Rush Current reduces installed cost and peak load. Reduces generator loads.

Lowest Noise - Only 70 dBa.

# Lowest Cost of Service Technology.

- Annually check electronics for tight connections
- No oil service or oil disposal EVER.



Replace capacitors every ten years

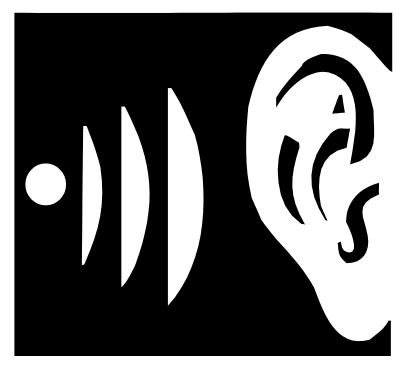
# **Lowest Noise and Vibration Levels**

• "Best in Class" Sound levels. No moving mechanical part touches any part of the housing or frame to transmit acoustic energy.

• Tested at 70 dBa at 1 meter with no sound attenuation.

• Vibration is essentially nonexistent.

 500 ton chiller operating at full speed produces 75 DB at 10 feet. Like a TV.



# **Flooded & DX Temperature Approach**

#### **Flooded Designs** With Shell and Tube



About 2° Approach

**Direct Expansion With Compact Brazed Plates** About 10° Approach

And Sand Jane Van

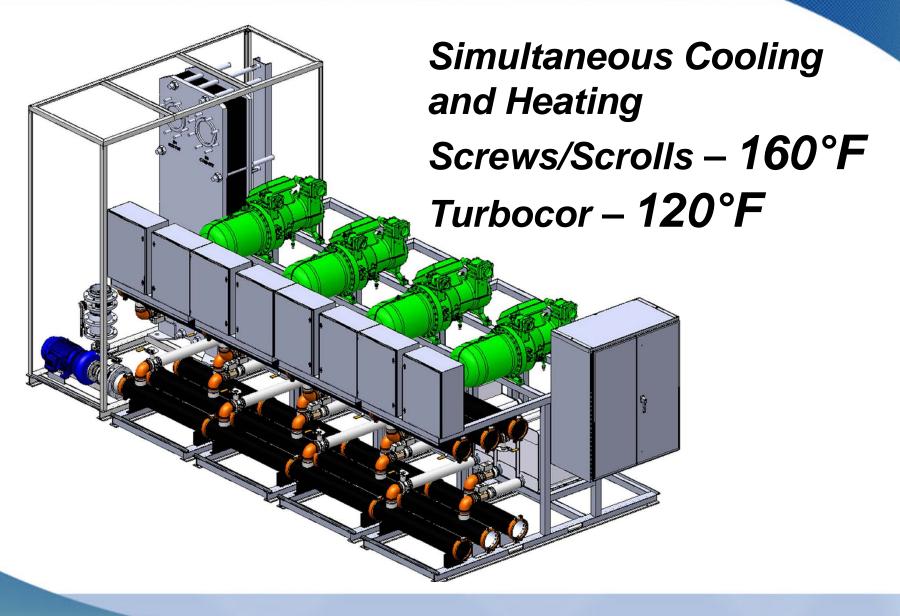
# **Range of Chillers Expanded Over Years**







# **Cooling and Heating Systems**



# **Chiller Only or Plant Control?**



Most manufacturers provide standard chiller controls that can operate cooling VSD tower fans and bypass valves. Perhaps resets.

Some manufacturers now offer <u>chiller mounted</u> Central Chiller Plant Energy Optimizing Controllers to directly correlate and optimize the entire chiller production around lowest total energy in <u>Open-Protocol</u> methods.

# TRIDUM



# **Native Tridium JACE Implications**

Arctic /	ACW 14	00		14-Dec-12	28:45:25 A	MEST	Syste	2m	APCTICC	
Setup	IO Data		Turbocors	Evaporat	or Conde	nsor Ala	arms Ti	rending 🖉	ARCTICCOOL	
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4	一般	0					Cit.		DEMAND	
	13	1		-1. <u>_</u>	2-24	김, 네가	le elle i		$\square$	
									0.0 %	
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									SPEED	
		-			_			-		
State		_			Offline State		(( 0.0 %))			
Setpoint Temp	45.0 °F	Evap	PSID	2.40 psi	Cond PSID	2.24 psi	Flow Proof	true		
Entering Evap T	68.3 °F	Ente	ring Cond T	68.9 °F	Pr	0.88	Evap Flow	1236.8 gal/min	CAPACITY	
Leaving Evap T	68.4 °F	Leav	ing Cond T	67.9 °F	Tons of R	-6.7 tR	Ourrent	A 0.0	CAPACITY	
Evap Delta T	-0.1 Δ°F	Cons	l Delta T	-1.0 Δ°F	EXV Pos	0.0 %	Power	0.0 kW	0.0%	
Te	75.0 F	Ť¢		68.2 °F	Cond Lvl	47.6 %	DTC Regid	0.0	0.048	
Evap Approach	-6.6 F	Cone	d Approach	0.2 °F	kW/TonR	0.00	96 FLA	0.0 %		
	INVE	RTER	SPEED	VANES	POWER	DEMAND S	UCTION	DISCHARGE	STATE	
COMPRESSO		1 °F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	71.7 psi	73.6 psi	Ready To Run	
COMPRESSOR	ALC: NO.	2 °F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	71.5 psi	73.1 psi	Ready To Run	
COMPRESSOR	THE REAL PROPERTY OF	L °F	0.0 rpm	0.0 %	0.0 kW		1.2 psi	74.4 psi	Ready To Run	
COMPRESSOR		2°F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	1.6 psi	74.6 psi	Ready To Run	
COMPRESSOR	15 78.	1 °F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	71.2 psi	74.0 psi	Ready To Run	
COMPRESSOR	R6 78.	2 °F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	1.7 psi	73.5 psi	Ready To Run	
		2 °F	0.0 rpm	0.0 %	0.0 kW	0.0 % 7	1.6 psi	74.5 psi	Ready To Run	

Powered by piagara AX FRAMEWORK®

Native systems run in "live-mode" on Tridium JACE hardware <u>without</u> Windows layer that slows execution rate and requires compiled code.

Controls contractors can easily connect to about anything.

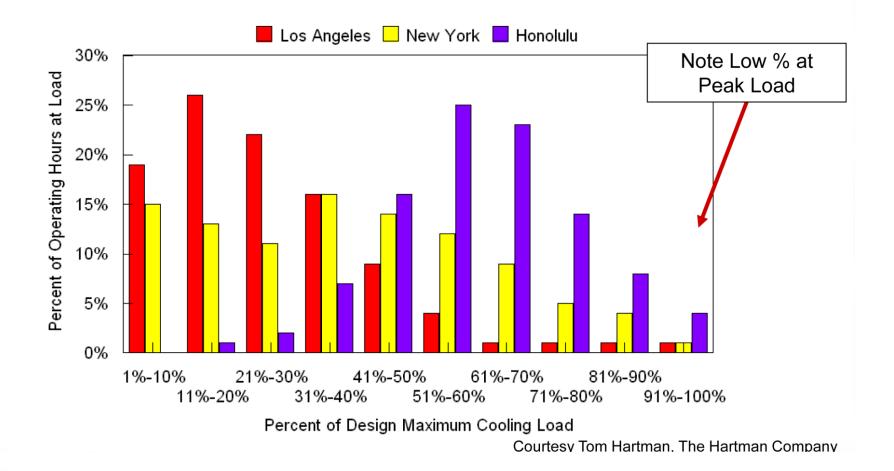
Large and growing array of drivers and algorithms from a network of developers.

# **Correlating Performance and Value**

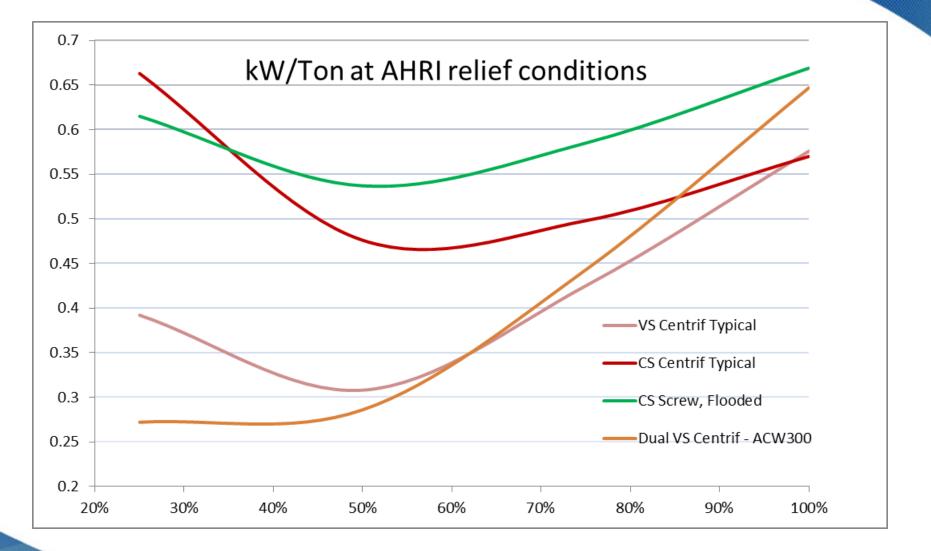
- **Determine the <u>Real Opportunity</u>** determine the actual condensing temperature possible water versus air cooled, evaporative condensing, adiabatic assist, sub-cooled design, free-cooling, rebates.
- Determine the <u>Actual Load Profile</u> to correlate real run hours at 10% load points.
- Correlate the <u>RIGHT Chiller Energy Maps</u> Based on actual load and real opportunity kW at lowest.
- <u>LCC Effects of OIL, Maintenance. Real Costs</u> Long-Term, Sustainability, Carbon and LEED needs.

## Load Profiles - Low Full Load Usage

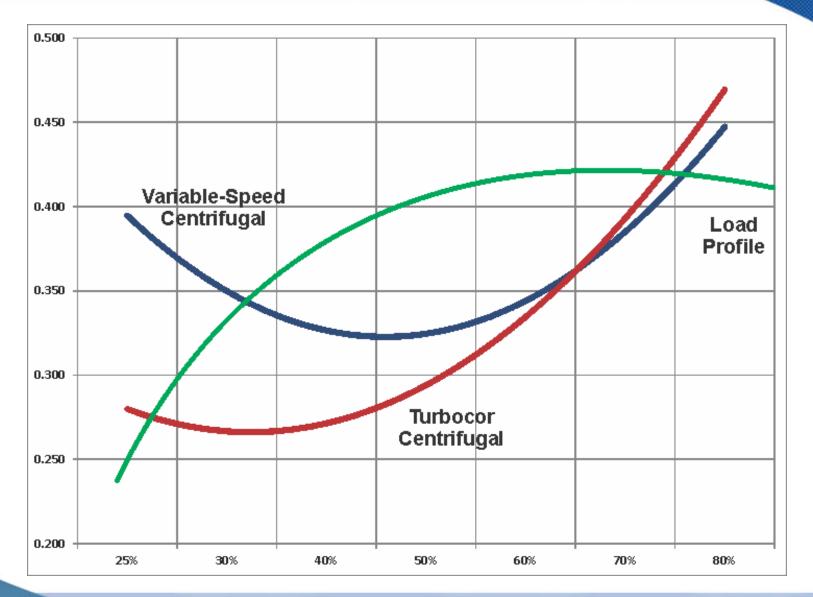
#### **Characteristics of Comfort Cooling Loads**



# **Load Profiles for Compressors**

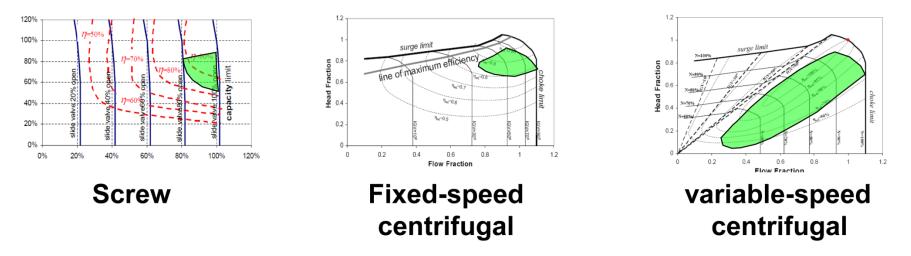


# **Energy With Actual Load Profile**



# **Efficiency Across a LARGER Envelope**

Map area with compressor efficiency within 10% of full-load compressor efficiency

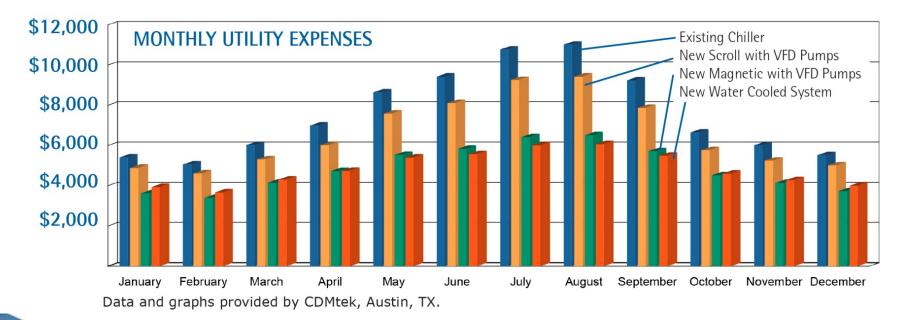


#### **Conclusion:**

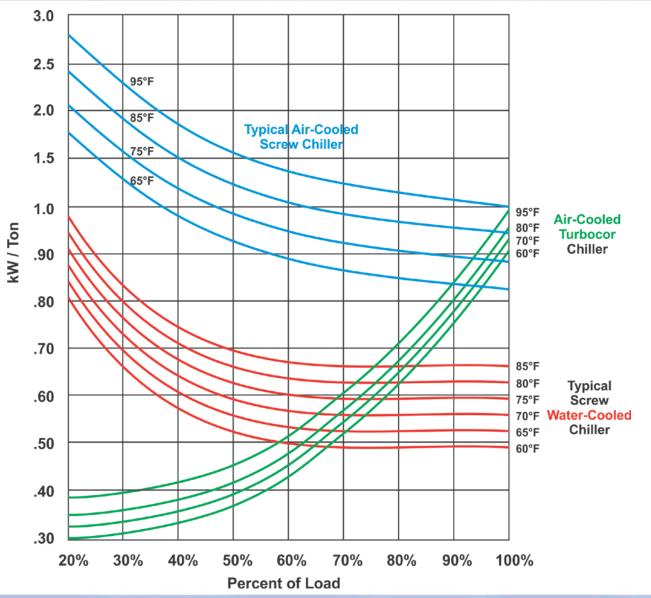
The variable-speed centrifugal compressor maintains peak efficiency over a much larger map area than either fixed speed centrifugal or screw compressors, explaining its excellent IPLV characteristics.

# **Load Profile and LCC Options**

PRODUCT ALTERNATIVES	FIRST-COST DIFFERENCE	ANNUAL SAVINGS	CUMULATIVE CASH FLOW DIFFERENCE	SIMPLE PAYBACK (Years)	INTERNAL RATE OF RETURN	LIFE CYCLE COST ADVANTAGE
New Scroll versus Existing	\$ 70,000	\$ 11,639	\$ 162,784	6.0	15.7%	\$ 44,275
New Magnetic versus Existing	\$ 163,000	\$ 32,462	\$ 486,249	5.0	19.3%	\$ 155,721
New Water Cooled versus Existing	\$ 258,000	\$ 32,706	\$ 396,119	7.9	11.1%	\$ 63,111
New Magnetic versus New Scroll	\$ 93,000	\$ 20,823	\$ 323,466	4.5	22.0%	\$111,446
Water Cooled versus New Magnetic	\$ 95,000	\$ 243	\$ -90,130	No payback	No payback	\$ -92,609



#### Water vs Air Cooled



# ASHRAE 90.1 – Path B Reality

#### 90.1, 2004 and the new 189.1, 2010 (Path B)

		Less than 150 Ton	150 to 300 Ton	300 to 600 Ton	> 600 Ton	
ASHRAE 189.1P,	Full Lood	5.500	5.500	5.862	5.961	СОР
Path (B), 2010	Full Load	0.639	0.639	0.600	0.590	kW/Ton
		7.816	7.816	8.792	8.792	СОР
	IPLV	0.450	0.450	0.400	0.400	kW/Ton
	Full Load	5.001	5.555	6.104	6.104	СОР
	Full Load	0.703	0.633	0.576	0.576	kW/Ton
ASHRAE 90.1, 2004		5.250	5.900	6.400	6.400	СОР
	IPLV	0.670	0.596	0.549	0.549	kW/Ton

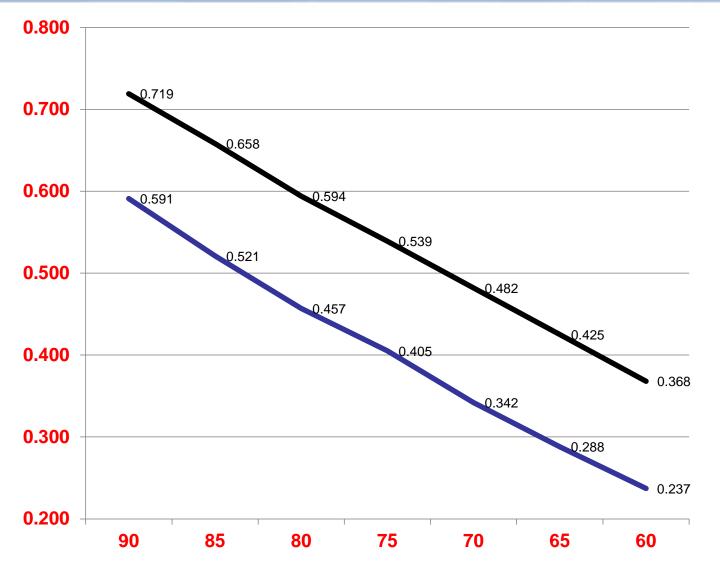
#### It's a Part-Load World

#### Implications for Path-A and Path-B Energy

- Constant-speed generally cannot meet Path-B.
- Two chillers may BOTH meet ASHRAE 90.1 BUT one may be 22% More Efficient on Path-B!

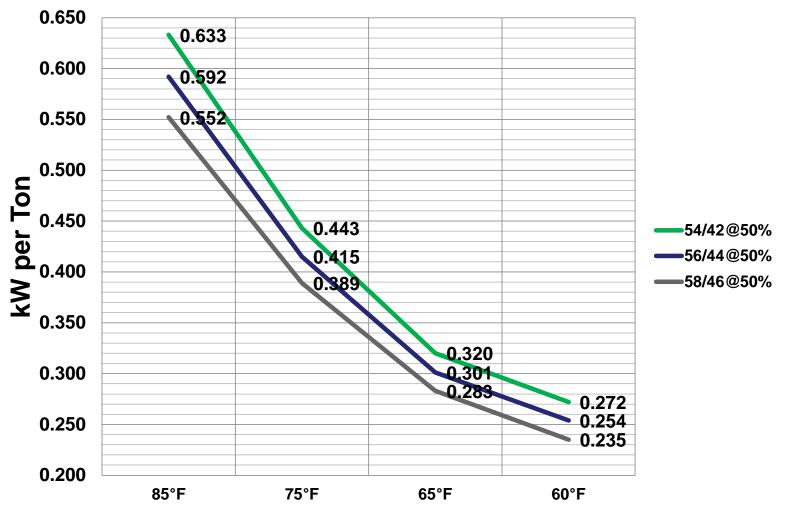
• Adding heat transfer surface and selecting correct full load point helps meet compliance.

#### 100% - vs - 50% Load

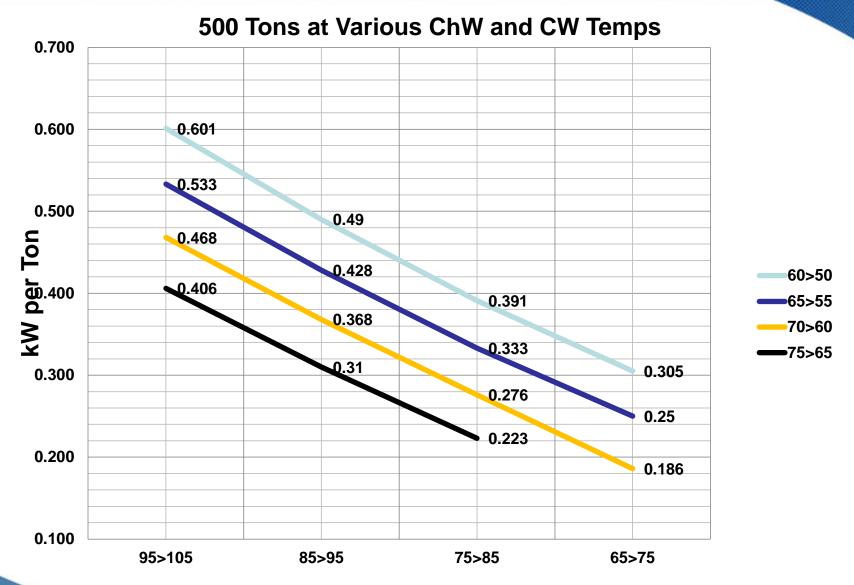


# **Energy for Turbocor Chillers**

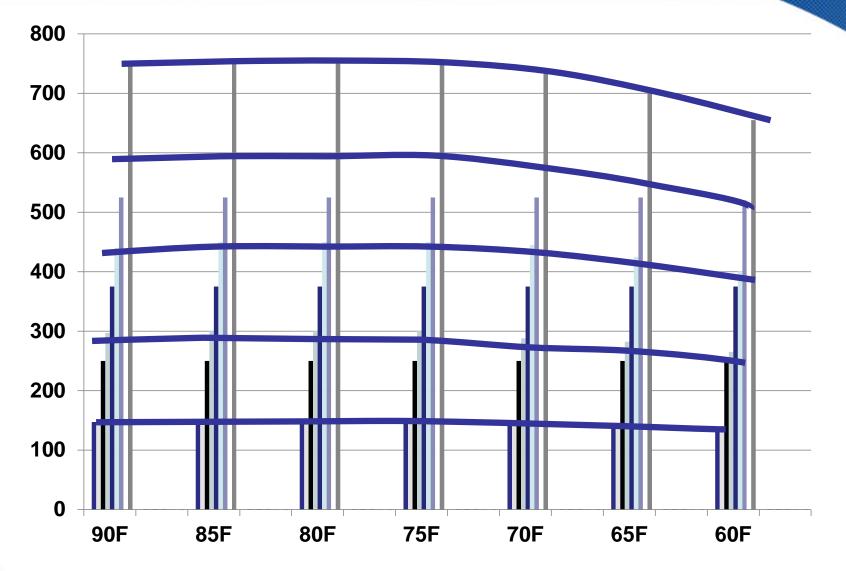
#### 50% Energy Performance - vs - Condenser Temp and Chilled Water Set Point



# **Energy Maps for Turbocor Chillers**



# **Capacity vs Condenser Temp**



# **Energy Cost Analysis**

# Houston St Louis Chicago Minneapolis



Typical of Hot, Warm and Cold Cities.

- Fan, pump, cooling tower fan, electric heating, lighting and miscellaneous equipment energy consumption are excluded.
- The results shown are for <u>chiller energy usage only</u>.
- Electrical costs at \$0.10 / kWh is shown.

# **Energy Analysis – Building Types**

# HLE –

- High internal load ratio
- Low occupancy (ASHRAE Std. Office Building)
- Economizer (airside)
- Typical Office use

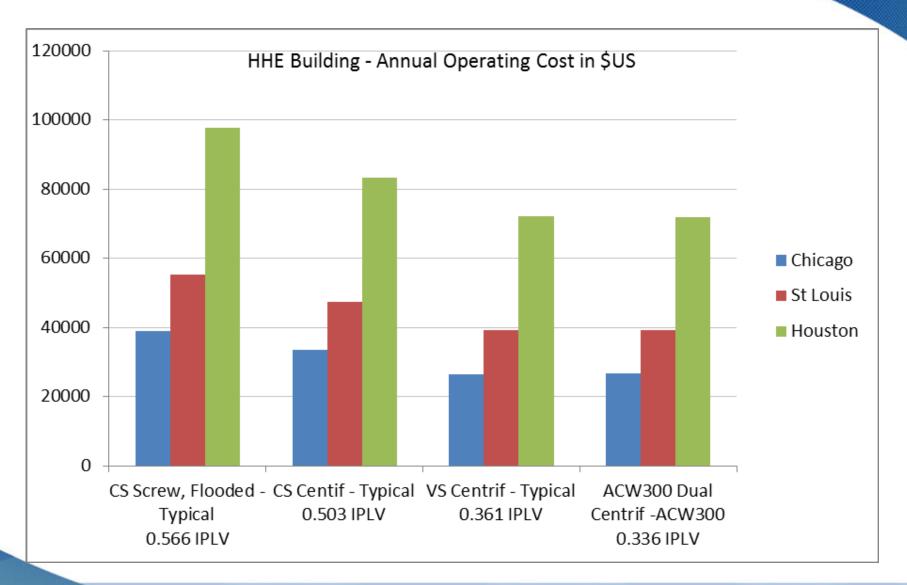
# HHE –

- High internal load ratio
- High occupancy 24/7 operation
- Economizer (airside)
- Typical Hospital use

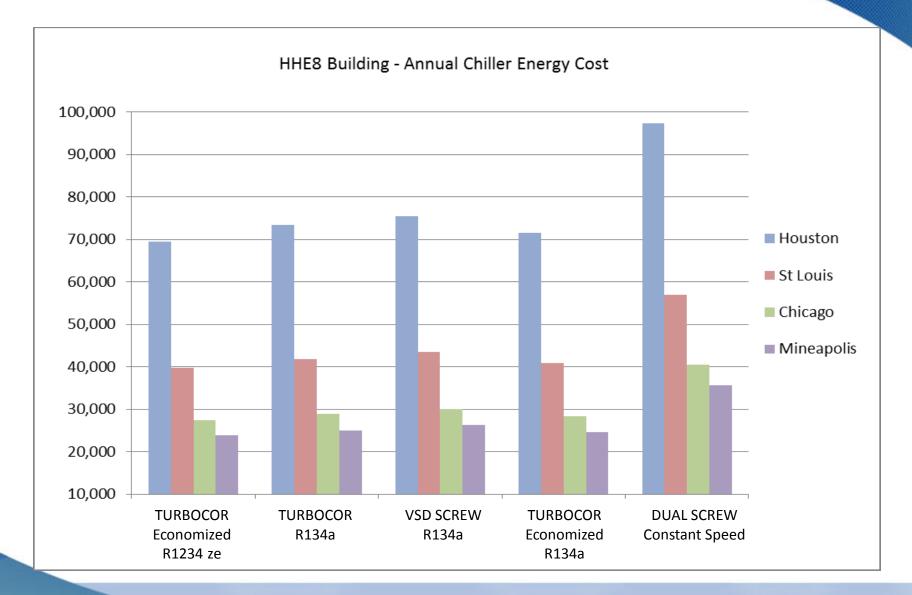
#### **ASHRAE Standard Office Building**

- Floor by floor VFD AHU with VAV electric reheat terminals
- Two equal parallel chillers with VSD pumps, VSD cooling tower fans
- Airside economizer option

# **Annual Energy Cost Comparison**



# **Energy Analysis Air-Cooled**



# **Energy Analysis Conclusions**

		Н	LE Buildi	ng	Н	ng	
		Chicago St Louis Houston C			Chicago	St Louis	Houston
CS Screw, Flooded - Typical	0.566 IPLV	0%	0%	0%	0%	0%	0%
CS Centif - Typical	0.503 IPLV	13%	14%	15%	14%	14%	15%
VS Centrif - Typical	0.361 IPLV	31%	29%	26%	32%	29%	26%
ACW300 Dual Centrif -ACW300	0.336 IPLV	32%	31%	28%	31%	29%	27%

Air Cooled Chillers - Annual Energy Cost at \$.10 per kW Hour - HHE8 Building								
		Houston		St. Louis		Chicago	Μ	inneapolis
Constant Speed Dual Screw	\$	97,000	\$	57,000	\$	40,000	\$	35,000
Variable Speed Dual Screw	\$	76,000	\$	43,000	\$	30,000	\$	27,000
Turbocor Oil Free	\$	73,000	\$	41,000	\$	29,000	\$	25,000
Turbocor Oil-Free with Economizer	\$	71,000	\$	40,500	\$	28,000	\$	24,000
Turbocor Oil-Free with Economizer and R1234ze	\$	69,000	\$	39,500	\$	27,000	\$	23,000
LLC - SUSTAINABLE - U-Value Effect of Oil, No Oil Maintenance, No Bearing Replacements								

While the values differ by application, the relative differences are consistent across building types, systems and locations. Variable speed control offers the greatest step change in kWh and operating cost reduction.

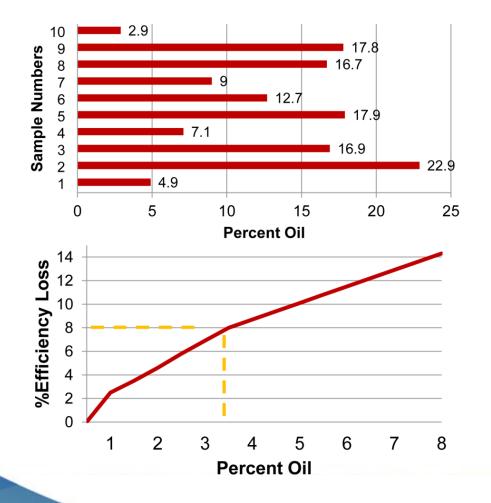
Oil-free makes it Sustainable and provides other LCC advantages.

# **High Cost of Oil - Maintenance**

Procedure	Daily	Weekly	Quarterly	Yearly
Record operating conditions (log)	X			
Check oil levels		X		
Check refrigerant levels		X		
Check oil return system			Х	
Check operation of motor starter			X	
Check sump heater and thermostat			X	
Inspect and adjust safety controls			X	
Leak check and repair leaks			Х	
Lubricate motor			Х	
Check and tighten electrical connections				X
Megohm motor windings				X
Perform oil analysis on lube oil				X
Replace oil filter and return filter/dryers				X
Replace or clean starter air filters				x

# **Oil Results in Huge Life-Cycle Costs**

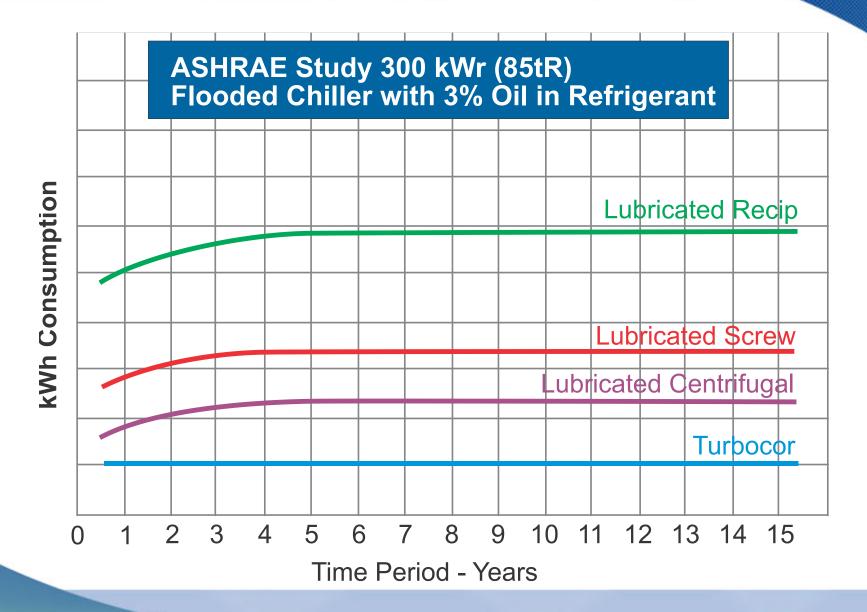
#### The Effect of Oil in the System



"An ASHRAE study determined that the vast majority of installed chillers have an excess amount of oil in the cooling system." ASHRAE research study 601

3.5 % of oil in the refrigerant charge reduces system efficiencies by 8 %

# **Oil Results in Huge Life-Cycle Costs**



# High Cost of Oil - Logarithmic Effect!

# BTUH = U \* A \* Log Mean Temp Difference

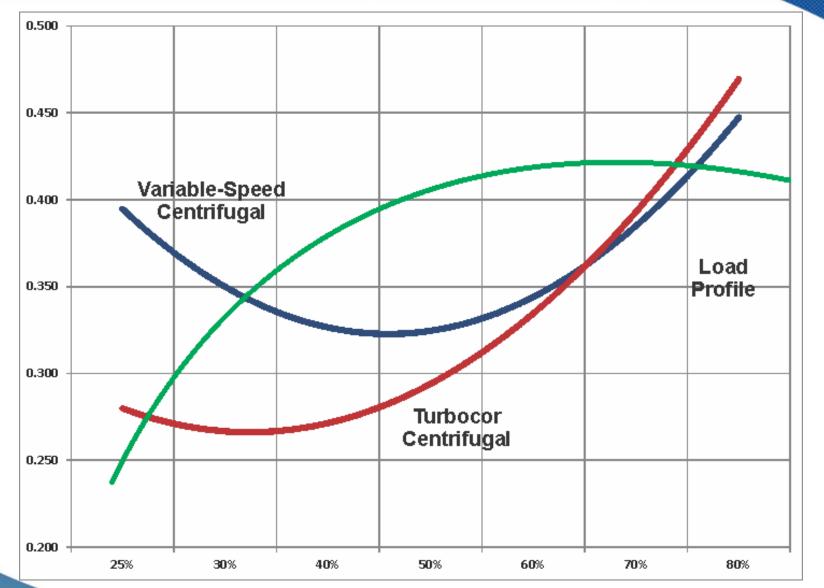
**U** - Heat Transfer Rate \* **A** - Surface Area \* **LMTD** 

Degrading the U-Value just 8% forces the compressor to INCREASE its energy to compensate for the loss.

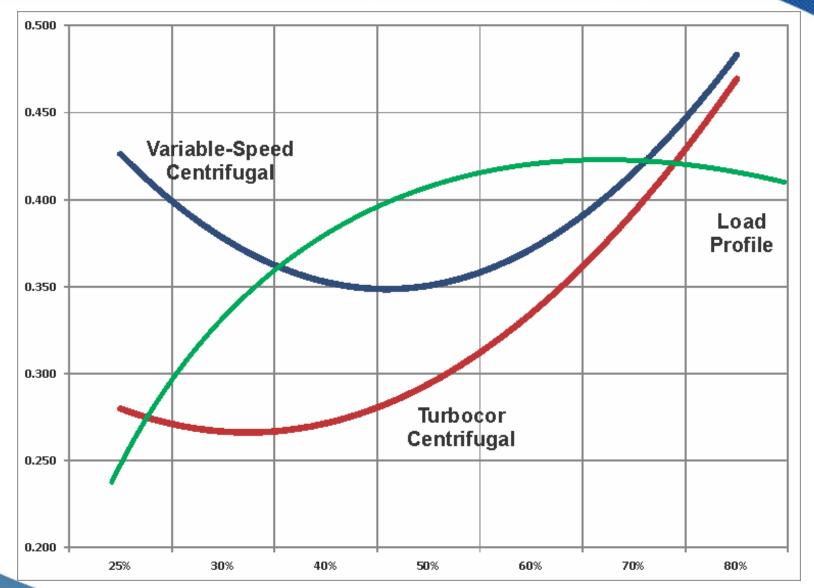
This is a logarithmic effect.

OIL IS NOT A SUSTAINABLE APPROACH!

# **Energy With Actual Load Profile**



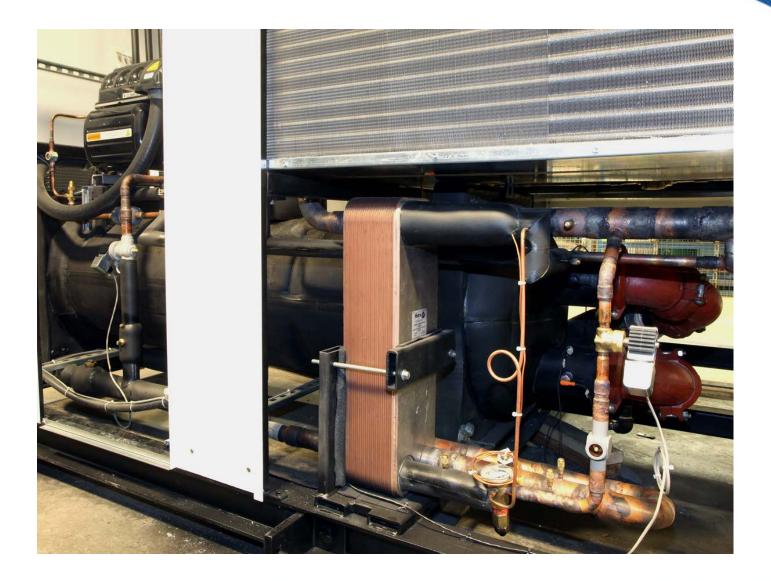
# **Energy – With 8% Loss from Oil**



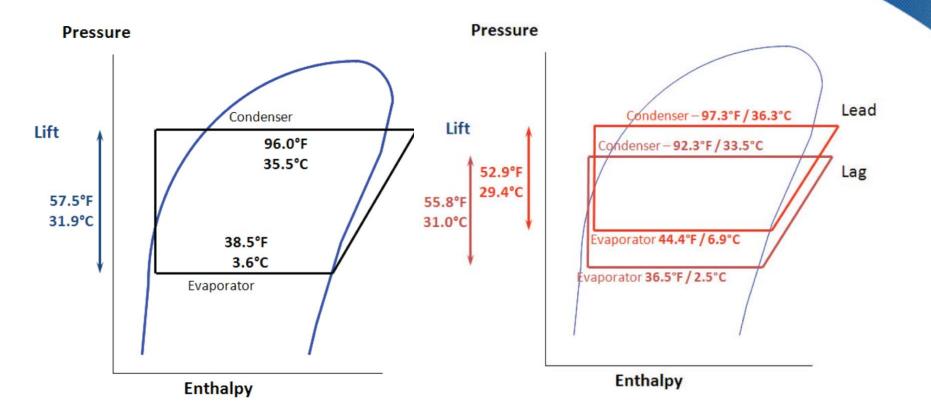
# **Key Things About Oil-Free Chillers**

- <u>Oil can Never be Sustainable</u> U-Value loss for the remaining service life.
- Oil causes energy to shift upward across the entire load. <u>Oil-free chillers</u> <u>have an effective payback at higher load points than when it was new.</u>
- Outside of energy, <u>oil-related service are the number one cost</u> of owning a chiller. <u>Lubricated part failure is the highest contributor to downtime.</u>
- There are <u>no compressor bearing inspections or periodic rebuilds</u>. There is one moving part in the compressor. Capacitor change every 10 years.
- There is <u>FAR less noise in magnetic bearings</u>. 70 dB without attenuation.
- <u>Service life may exceed fifty years with sustainable energy efficiency.</u>
- The <u>Reduced Carbon-Footprint</u> of operating at lower energy is seen as a responsible environmental approach.

# **Economizers Reduce Energy at Full Load**



# **Series-Counter-Flow Reduces Lift**



Chillers in parallel requires each to produce the total system lift. Series counter-flow reduces the lift requirement for both chillers.

# **Air Cooled Evap-Assist Condensing**

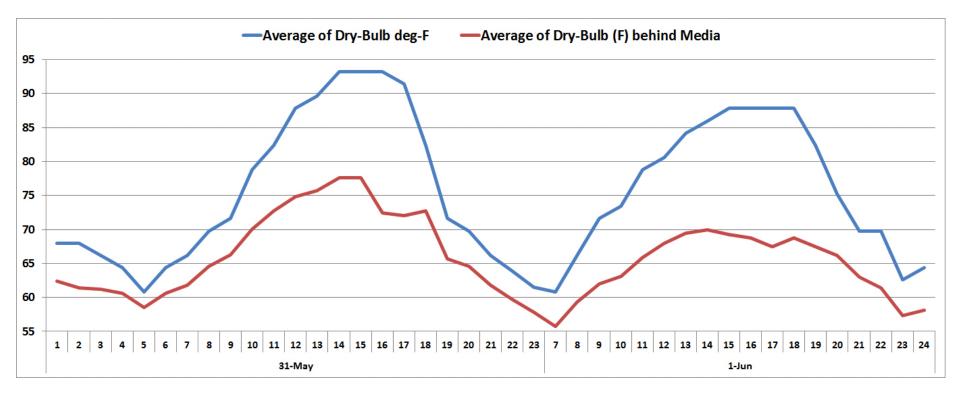






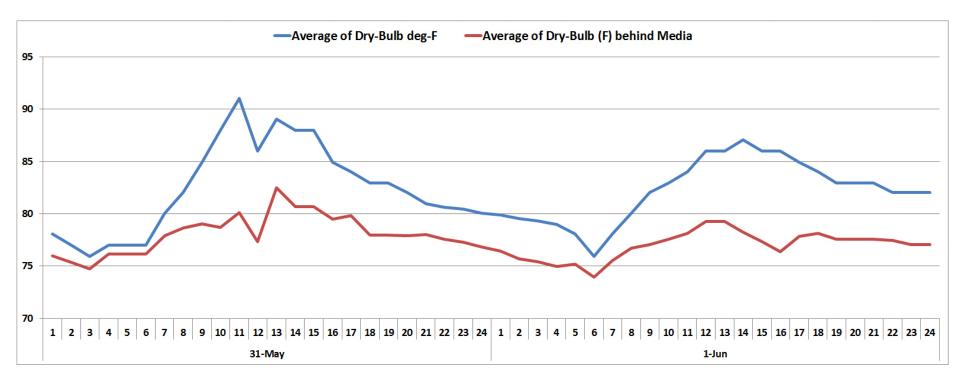
# **Air Cooled Evap-Assist Condensing**

# **Sacramento CA**



# **Air Cooled Evap-Assist Condensing**

# **Miami FL**



# Air Cooled Evap-Assist Condensing Las Vegas Result

Hours above  $45^{\circ}F - 6,302$  Hours  $- 22.29^{\circ}$  Avg WBD Hours Above  $60^{\circ}F - 2,716$  Hours  $- 28.13^{\circ}$  Avg WBD Average Utility Rate - \$0.15 - Ambient  $105^{\circ}F$ . Average Wet-Bulb Depression Effect =  $20^{\circ}F$ 

3,000 Hours x \$0.15 kW/Hr x 1.022 kW/ton = \$114,975 3,000 Hours x \$0.15 kW/Hr x .693 kW/ton = \$77,963

Annual Energy Only Savings = <u>\$37,012</u>

Requires no cooling tower, water treatment, water make-up or condenser pumps

# **Low-Noise and Higher Efficiency**





Figure 1: Using a diffuser greatly reduces the losses experienced once the impeller has discharged the air

## **Fluid Coolers For Pre and Free Cooling**



**Energy** – Correlated with <u>True CW temps and</u> Load Profiles and Optimized Chiller Loading **Utility Rebates** – Can Pay for Premium Cost Maintenance – No Oil, Rebuilds, Lowest Cost **U-Value Degradation** – <u>8%</u> is common loss Lowest Noise - Technologies Available **Air Cooled** versus Water Cooled

Oil-Free is the ONLY Sustainable Low Energy Chiller Approach

# **The Bottom Line About Creating Value**

The Economic Opportunity May be Compelling...

**Better Technology** Can Promote Change...

**Regulations & Standards** Can Speed Change...

**Environmental Responsibility Suggest Change...** 

# **Total Cost of Ownership Powers Change**

#### **Questions or Comments...**

Questions?



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